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H-Adaptive Finite Element Methods for 1D Stationary High Gradient Boundary Value Problems



Collins Olusola Akeremale^{1,2*}, Oluwasegun Adeyemi Olaiju^{1,3}, Su Hoe Yeak¹

¹ Department of Mathematical Sciences, Universiti Teknologi Malaysia, Skudai 81310, Johor Bahru, Malaysia

² Department of Mathematics, Federal University Lafia, PMB 146, Nigeria

³ Department of Mathematics and Statistics, Federal Polytechnic, Ilaro, PMB 50, Nigeria

Corresponding Author Email: ocakeremale@graduate.utm.my

https://doi.org/10.18280/mmep.080617	ABSTRACT
Received: 15 June 2021	This article considered the traditional finite element method (FEM) and adaptive finite
Accepted: 8 October 2021	element method (FEM) for the numerical solution of the one-dimensional boundary
≜	value problems. We established the preference or the superiority of the h-adaptive FEM
Keywords:	to traditional FEM in high gradient problems in terms of accuracy and cost of
adaptivity, advection, fine region, finite element	computation. Numerical examples which confirm the performance and adaptability of
method, high-gradient	the h-adaptive method over the traditional finite element method and the high accuracy
meinou, nign-grauieni	of the numerical solution are presented. Detailed error analysis of linear elements was

with high gradient problems.

1. INTRODUCTION

Most natural phenomena are characterized by high gradients where a rapid change in the solution of the model occurs. Using global or uniform refinement may resolve the issue at a very high computational cost and time since these high gradients are normally localised [1, 2]. It is essential to accurately solve such models (problems) as it helps to make appropriate decisions and to predict future occurrences. Though there are different numerical methods for the solution of such models (differential equations) [3, 4] in this article, the finite element method is adopted because it is efficient, accurate, easy to code and able to handle irregular geometry [5]. The finite element method is one of the numerical approaches to boundary value problems. The method concerns partitioning the domain of the solution into a finite number of essential subdomains and using the variational approach to create an estimate of the solution over the set of finite elements. In general, the underlying idea of this method had been used with impressive accomplishment in solving problems, virtually in all fields of mathematical physics and engineering [6, 7].

The (FEM) has been used extensively to approximate the solution of partial differential equations [8, 9]. To give a smooth solution, the FEM has been improved upon in many ways. However, applications of this technique to high gradient boundary value problems are not trivial. For high gradients and singularities problems, mesh refinement is needed.

Moreover, for any discontinuous or high gradient problem, uniform refinement may be extremely expensive for these classes of problems. This is because many elements that are not in the high gradient region and do not require refinement will be refined thereby resulting in serious oscillation in the solution [10-12]. Thus, using local refinement that selectively refines only the elements within the high gradient region seems to be a better choice [13-16]. The adaptive finite element method [9, 17-19] has the potential to overcome this problem and produce accurate results without refining the mesh of the whole domain.

also discussed. In conclusion, h-adaptive FEM is recommended for complex systems

High gradients, as often found in engineering applications, such as fluid, turbulence, shocks, often affect the optimal performance, tolerance, and physical behaviour of such applications. Therefore, effective, and efficient advanced numerical methods in modelling such behaviours are essential in the scientific community [19, 20].

This article employs the adaptive FEM to approximate the high gradient problems to establish the superiority of the adaptive FEM over FEM. It was observed from the given examples in this work that adaptive finite element performs better in term accuracy and computational time compared to traditional finite element method. In section two, the methodology is discussed, section three discusses the results and finally, section four discusses the conclusion and recommendations. Figure 1 and Figure 2 below shows a linear adaptive geometry and geometry with six degrees of polynomial without adaptive respectively



Figure 1. A linear polynomial adaptive FE geometry for second elements (h-adaptive)



Figure 2. Galerkin FE with six degrees of polynomial interpolation in each element

Consider the general steady one-dimensional second-order ordinary differential advection-dominated equation with two boundary conditions below:

$$D\frac{\partial^2 u}{\partial x^2} - v\frac{\partial u}{\partial x} + c(x)u(x) = f(x), \qquad 0 < x < 1$$
(1)

where, v and D are the known constant velocity (advection) and diffusivity, respectively.

With the boundary conditions $u(0)=u_0$, $u(L)=u_n$, where u is an unknown scalar function of x on the interval $[x_L, x_R]$, f and c are given functions of x. The function f(x) is the known source function.

2. METHODOLOGY

The finite element method (FEM) is a numerical approach for partial differential equations with boundary conditions. It subdivides the domain of the problem into simpler parts, called elements, and adopts a residual approach for the solution of the problem through minimization of the error function associated with the discretized problem [6, 7].

The main task in the finite element method is finding approximate solution u such that.

$$u^e = \sum_{i=1}^n N_i u_i$$

where, *n* is the total number of nodes in the domain and N_i is the basis function corresponding to the nodes in each element, as shown in Figure 3. A system of equations is generated and solved. The solution of FEM in its weak form is C^0 continuous, and it includes boundary conditions in its formulation.

Eq. (1) is converted to weak form by integration over the domain as:

$$\int_{\Omega} \left[D \frac{\partial^2 u}{\partial x^2} - v \frac{\partial u}{\partial x} + c(x)u(x) \right] dx = \int_{\Omega} (w \cdot f) \, dx \tag{2}$$

Eq. (2) becomes,

$$w \cdot D \frac{\partial u}{\partial x} \Big|_{0}^{L} - \int_{0}^{L} w \cdot v \frac{\partial u}{\partial x} dx = \int_{0}^{L} w \cdot f dx - \int_{0}^{L} w \cdot c(x) u(x) dx$$
(3)

where, w is any weight function.

Let, w(0)=0, and $\frac{\partial u}{\partial x}=u_n$, we get from Eq. (3),

$$w(L) \cdot D \cdot u_n - \int_0^L D \frac{\partial w}{\partial x} \frac{\partial u}{\partial x} dx - \int_0^L w \cdot v \frac{\partial u}{\partial x}$$

= $\int_0^L w \cdot f dx - \int_0^L w \cdot c(x) u(x) dx$ (4)

$$w(L) \cdot D \cdot u_n - \sum_{i=1}^{n-1} \int_{x_i}^{x_{i-1}} D \frac{\partial w}{\partial x} \frac{\partial u}{\partial x} dx$$
$$- \sum_{i=1}^{n-1} \int_{x_i}^{x_{i-1}} w \cdot v \frac{\partial w}{\partial x} dx \qquad (5)$$
$$= \sum_{i=1}^{n-1} \int_{x_i}^{x_{i-1}} w \cdot f \, dx - \sum_{i=1}^{n-1} \int_{x_i}^{x_{i-1}} w \cdot c(x) u(x) \, dx$$

Let,

$$u_a = \sum_{j}^{n} N_j(x) u_j, w = \sum_{i}^{n} N_i \phi_i$$

where, u_a is the approximate solution, u_j are the unknown nodal values and N_i , N_j are the essential functions needed to obtain the approximate solution. w is the weight function, and it is a scalar quantity, ϕ is a virtual constant.

2.1 Shape function

In the study of the finite element method, the basis function is the connection between nodes while discretizing the domain problem. Also, the basis function must comply with the continuity conditions associated with the underlying PDE and the finite element formulation. The basis functions must satisfy the following properties:

2.1.1 Partition of unity

$$\sum_{k=1}^{n} N_k = 1 \tag{6}$$

2.1.2 Kronecker delta property

$$N_i(x_k) = \delta_{ik} \tag{7}$$

Linear shape functions for linear elements are given as:

$$N = \left[\frac{1-\xi}{2}, \frac{1+\xi}{2}\right].$$

Similarly, we get,

$$x = [N_1, N_2] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}, \frac{\partial x}{\partial \xi} = \frac{x_1 - x_2}{2}, \frac{\partial x}{\partial \xi} = \frac{l}{2} d\xi.$$
(8)

Therefore, Eq. (5) can be expanded as:

$$w(L) \cdot D \cdot u_n - \sum_e \frac{l_e}{2} \int_{-1}^{1} D \frac{\partial w}{\partial x} \frac{\partial u^e}{\partial x} d\xi$$
$$- \sum_e \frac{l_e}{2} \int_{-1}^{1} w \cdot v \frac{\partial u^e}{\partial x}$$
(9)
$$= \sum_e \frac{l_e}{2} \int_{-1}^{1} w \cdot f \, d\xi + \sum_e \frac{l_e}{2} \int_{-1}^{1} w \cdot c(x) u(x) d\xi$$

Using the general property of matrix Transpose, we have,

$$\frac{\partial w}{\partial x}\frac{\partial u}{\partial x} = (B_T\phi)(B_Tu^e) = (B_T\phi)^T(B_Tu^e),$$
$$= \phi^T(B_T^TB_T)u^e$$

where,

$$B_{T} = \frac{\partial N}{\partial x},$$

$$w \cdot v \frac{\partial u}{\partial x} = \phi \cdot v(\xi) B_{T} u^{e},$$

$$w \cdot v \frac{\partial u}{\partial x} = \phi^{T} v(\xi) B_{T} u^{e}$$
(10)

$$w(L) \cdot D \cdot u_n - \sum_{i=1}^{n-1} \int_{x_i}^{x_{i+1}} D \frac{\partial w}{\partial x} \frac{\partial u}{\partial x} dx - \sum_{i=1}^{n-1} \int_{x_i}^{x_{i+1}} w \cdot v \frac{\partial u}{\partial x} dx$$
$$= \sum_{i=1}^{n-1} \int_{x_i}^{x_{i+1}} (w \cdot f) dx - \sum_{i=1}^{n-1} \int_{x_i}^{x_{i+1}} w \cdot c(x) u(x) dx \cdot c(x) u(x) dx.$$

After re-arrangement, we have,

$$w(L) \cdot D \cdot u_{n} - \sum_{e} \frac{l_{e}}{2} \phi^{T} \int_{-1}^{1} D \cdot (B_{T}^{T} B_{T}) u^{e} d\xi - \sum_{e} \frac{l_{e}}{2} \phi^{T} \int_{-1}^{1} N^{T} \cdot v \cdot B_{T} u^{e} d\xi - \sum_{e} \frac{l_{e}}{2} \phi^{T} \int_{-1}^{1} N^{T} f(\xi) d\xi - \sum_{e} \frac{l_{e}}{2} \phi^{T} \int_{-1}^{1} c(\xi) N^{T} N u^{e} d\xi = 0$$
(11)

$$w(L) \cdot D \cdot u_{n} - \sum_{e} \frac{l_{e}}{2} \phi^{T} \int_{-1}^{1} (D(B_{T}^{T}B_{T}) + N^{T}vB_{T} + c(\xi)N^{T}N) u^{e}d\xi$$
(12)
$$- \sum_{e} \frac{l_{e}}{2} \int_{-1}^{1} \phi^{T}N^{T} d\xi = 0$$

$$-\sum_{e} \frac{l_{e}}{2} \phi^{T} \int_{-1}^{1} (D(B_{T}^{T}B_{T}) + N^{T} v B_{T} + c(\xi) N^{T} N) u^{e} d\xi$$

$$= \sum_{e} \frac{l_{e}}{2} \int_{-1}^{1} \phi^{T} N^{T} d\xi - w(L) \cdot D \cdot u_{n}$$
(13)

The integral equation above, upon evaluation, gives a linear system of algebraic equations for n unknowns of the problem.

3. RESULTS AND DISCUSSION

In this section, two high gradient problems are considered, and their results are presented which show the superiority of the adaptive finite element method over the traditional finite element method.

Example 1.

Given the following exact solution (refer to Eq. (1)),

$$u = tan^{-1} \left(\frac{x - 0.5}{\alpha}\right), f = \frac{2\alpha(x - 0.5)}{(\alpha^2 + (x - 0.5)^2)^2},$$

$$D = 1, v = 0, c = 0, \alpha = 0.01.$$

$$u(0) = tan^{-1} \left(-\frac{0.5}{\alpha}\right), u(1) = tan^{-1} \left(\frac{0.5}{\alpha}\right).$$

Table 1 shows the result of the analysis. From this table, it takes forty-eight elements for traditional FEM to arrive at an error value of 0.042372 while it takes h-adaptive FEM, forty-eight elements with four parental elements to arrive at an error value of 0.007007. It clearly shows that h-adaptive FEM is better compared to traditional FEM. Figures 3 and 4 shows the error graph of adaptive FEM and traditional linear FEM respectively. Figure 5 and Figure 6 shows the comparison of the exact solution and the numerical solution for the adaptive FEM and traditional FEM respectively.

Table 1. Comparison of the error of the two methods



Figure 3. *h*-adaptive FEM error



Figure 4. Linear FEM error



Figure 5. *h*-adaptive FEM solution



Figure 6. Linear FEM solution

Example 2.

Given the following exact solution (refer to Eq. (1)),

$$T = (1 - x) (\tan^{-1} (k(x - x_0)) + \tan^{-1} (kx_0)),$$

$$T(0)=0, T(1)=1, v=0, c=0, D = \frac{1}{a} + a(x - x_0)$$

$$f = 2\{1 + k(x - x_0)[\tan^{-1} (x - x_0) + \tan^{-1} (kx_0)]\},$$

Both the *h*-adaptive finite element method and the traditional finite method are also considered for this question to compare their error. Table 2 shows the result of the analysis, taking k=50 and $x_0=0.5$.

Table 2. Comparison of the error of the two methods



Figure 7. *h*-adaptive FEM error



Figure 8. Linear FEM error



Figure 9. *h*-adaptive FEM solution



Figure 10. Linear FEM solution (*h*=1/20)

Table 2 above shows that it takes forty-four elements for traditional FEM to arrive at an error value of 0.017763 while it takes h-adaptive FEM, forty-eight elements with four parental elements to arrive at an error value of 0.001740. It clearly shows that h-adaptive FEM is better compared to traditional FEM. Figures 7 and 8 above shows the error graph of adaptive FEM and traditional linear FEM respectively. Figure 9 and Figure 10 above shows the comparison of the exact solution and the numerical solution for the adaptive FEM and traditional FEM respectively.

3.1 Error analysis

This section focuses on the truncation error of the displacement. The error of the finite element method with constant mesh is considered. Truncation error through Taylor series expansion is expressed as shown below.

From Eq. (3) above, we have,

$$w \cdot D \frac{\partial u}{\partial x} \Big|_{0}^{L} - \int_{0}^{L} D \frac{\partial w}{\partial x} \frac{\partial u}{\partial x} dx - \int_{0}^{L} w \cdot v \frac{\partial u}{\partial x} dx + \int_{0}^{L} w \cdot c(x)u(x) dx = \int_{0}^{L} (w \cdot f) dx$$
(14)

substitute for:

$$u = u_h + \frac{h^2}{2!} u^{\prime\prime}(x^*), u^\prime = u_h^\prime + \frac{h^2}{2!} u^{\prime\prime\prime}(x^*), \qquad x^* \in (0,L)$$

in Eq. (14) above gives,

$$w \cdot D\left(\frac{\partial}{\partial x}u_{h} + \frac{h^{2}}{2}u'''(x^{*})\right)\Big|_{0}^{L} - \int_{0}^{L}D \cdot \frac{\partial w}{\partial x}$$
$$\cdot \left(\frac{\partial}{\partial x}u_{h} + \frac{h^{2}}{2}u'''(x^{*})\right)dx$$
$$- \int_{0}^{L}w \cdot v \cdot \left(\frac{\partial}{\partial x}u_{h} + \frac{h^{2}}{2}u'''(x^{*})\right)dx \quad (15)$$
$$+ \int_{0}^{L}w \cdot c(x) \cdot \left(u_{h} + \frac{h^{2}}{2}u'''(x^{*})\right)dx$$
$$= \int_{0}^{L}(w \cdot f)dx$$
$$w = N\phi \equiv N_{i}, \quad i \in IN, \quad w(0) = 0$$

where, IN represents internal and Neuman boundary nodes.

$$w(L) \cdot D \cdot \frac{\partial u_h}{\partial x}\Big|_0^L + w(L) \cdot D \cdot \frac{h^2}{2} u^{\prime\prime\prime}(x^*)$$

$$-\int_{0}^{L} D \cdot \frac{\partial w}{\partial x} \frac{\partial u_{h}}{\partial x} dx - \int_{0}^{L} D \cdot \frac{\partial w}{\partial x} \cdot \frac{h^{2}}{2} u^{\prime\prime\prime}(x^{*}) dx$$
$$-\int_{0}^{L} w \cdot v \frac{\partial u_{h}}{\partial x} dx - \int_{0}^{L} w \cdot v \cdot \frac{h^{2}}{2} u^{\prime\prime\prime}(x^{*}) dx$$
$$+\int_{0}^{L} w \cdot c(x) \cdot u_{h} dx + \int_{0}^{L} w \cdot c(x) \cdot \frac{h^{2}}{2} u^{\prime\prime}(x^{*}) dx = \int_{0}^{L} (w \cdot f) dx$$

Re-arrange, we have,

$$-\int_{0}^{L} D \cdot \frac{\partial N_{i}}{\partial x} \cdot \frac{\partial u_{h}}{\partial x} dx - \int_{0}^{L} N_{i} \cdot v \frac{\partial u_{h}}{\partial x} dx + \int_{0}^{L} N_{i} \cdot c(x) u_{h} dx$$
$$= \int_{0}^{L} N_{i} \cdot f dx + \int_{0}^{L} D \cdot \frac{\partial N_{i}}{\partial x} \cdot \frac{h^{2}}{2} u^{\prime\prime\prime}(x^{*}) dx$$
$$+ \int_{0}^{L} N_{i} \cdot v \cdot \frac{h^{2}}{2} u^{\prime\prime\prime}(x^{*}) dx - \int_{0}^{L} N_{i} \cdot c(x) \cdot \frac{h^{2}}{2} u^{\prime\prime\prime}(x^{*}) dx$$
$$- N_{i}(L) \cdot D \cdot \frac{\partial u_{h}}{\partial x} \Big|_{L} - N_{i}(L) \cdot D \cdot \frac{h^{2}}{2} u^{\prime\prime\prime\prime}(x^{*})$$

$$-\sum_{j}^{n} \left(\int_{0}^{L} D \cdot \frac{\partial N_{i}}{\partial x} \cdot \frac{\partial N_{j}}{\partial x} dx \right) u_{j} - \sum_{j}^{n} \left(\int_{0}^{L} N_{i} \cdot v \cdot \frac{\partial N_{j}}{\partial x} dx \right) u_{j}$$
$$+ \sum_{j}^{n} \left(\int_{0}^{L} N_{i} \cdot c(x) \cdot N_{j} dx \right) u_{j}$$
$$= \int_{0}^{L} N_{i} \cdot f dx + \int_{0}^{L} D \cdot \frac{\partial N_{i}}{\partial x} \cdot \frac{h^{2}}{2} u'''(x^{*}) dx$$
$$+ \int_{0}^{L} N_{i} \cdot v \cdot \frac{h^{2}}{2} u'''(x^{*}) dx$$
$$- \int_{0}^{L} N_{i} \cdot c(x) \cdot \frac{h^{2}}{2} u''(x^{*}) dx - N_{i}(L) \cdot D$$
$$\cdot \frac{\partial u_{h}}{\partial x} \Big|_{L} - N_{i}(L) \cdot D \cdot \frac{h^{2}}{2} u'''(x^{*})$$

Let

$$D_{ij} = \int_0^L D \cdot \frac{\partial N_i}{\partial x} \cdot \frac{\partial N_i}{\partial x} dx, \qquad v_{ij} = \int_0^L N_i \cdot v \cdot \frac{\partial N_j}{\partial x} dx$$
$$c_{ij} = \int_0^L N_i \cdot c(x) N_j dx, \qquad k_{ij} = -D_{ij} - v_{ij} + c_{ij},$$

We have,

$$\sum_{j=1}^{n} k_{ij} u_{j} = \int_{0}^{L} N_{i} \cdot f dx + \frac{Dh^{2}}{2} u'''(x^{*}) (N_{i}(L)) + v$$

$$\cdot \frac{h^{2}}{2} \int_{0}^{L} u'''(x^{*}) N_{i} dx$$

$$- \frac{h^{2}}{2} \int_{0}^{L} u''(x^{*}) N_{i} \cdot c(x) dx$$

$$- N_{i}(L) \cdot D \cdot \frac{\partial u_{h}}{\partial x} \Big|_{L}$$

$$- N_{i}(L) \cdot D \cdot \frac{h^{2}}{2} u'''(x^{*})$$

$$N_{a} = [N_{i}|_{0}^{L}] = N_{i}(L), \qquad N_{b} = \left[\int_{0}^{L} N_{i} \cdot f dx \right],$$
(16)

$$N_c = \left[\int_o^L u''(x^*)N_i \cdot c(x)dx\right], \qquad i \in IN(i \neq 0)$$

Therefore,

$$N_{a} = \begin{bmatrix} 0 \\ \vdots \\ 1 \end{bmatrix}$$

$$Ku = f + \frac{Dh^{2}}{2}u'''(x^{*})\begin{bmatrix} 0 \\ \vdots \\ 1 \end{bmatrix} + v\frac{h^{2}}{2}u'''(x^{*})N_{b} - \frac{h^{2}}{2}N_{c}$$

$$-\frac{Dh^{2}}{2}\begin{bmatrix} 0 \\ \vdots \\ u'''(x^{*}) \end{bmatrix}$$
(17)

 $Ku = f + \Delta f$

where,

$$\Delta f = \frac{Dh^2}{2} u^{\prime\prime\prime}(x^*) \begin{bmatrix} 0\\ \vdots\\ 1 \end{bmatrix} + v \frac{h^2}{2} u^{\prime\prime\prime}(x^*) N_b - \frac{h^2}{2} N_c - D \frac{h^2}{2} \begin{bmatrix} 0\\ \vdots\\ u^{\prime\prime\prime}(x^*) \end{bmatrix}$$
(18)

$$\left| |\Delta f| \right| \le h^2 |u^{\prime\prime\prime}(x^*)| \left(D + \frac{\nu}{2} \left| |N_b| \right| \right) + \frac{h^2}{2} |u^{\prime\prime}(x^*)| \left| |N_c| \right|$$
(19)

Theorem 1:

Given a linear system Ax=b where $A \in \mathbb{R}^{n \times n}$, $b \in \mathbb{R}^{n}$ and $x \in \mathbb{R}^{n}$ With condition number given as k(A). Let $\delta b \in \mathbb{R}^{n}$ be a small perturbation of *b* and define $x + \delta x \in \mathbb{R}^{n}$ as the solution of the system:

$$A(x + \delta x) = (b + \delta b),$$

Then,

$$\frac{\left|\left|\delta x\right|\right|}{\left|\left|x\right|\right|} \le k(A) \le \frac{\left|\left|\delta b\right|\right|}{\left|\left|b\right|\right|}$$

This can be written as,

$$\frac{\left|\left|\delta b\right|\right|}{k(A)\left|\left|b\right|\right|} \le \frac{\left|\left|\delta x\right|\right|}{\left|\left|x\right|\right|} \le \frac{k(A)\left|\left|\delta b\right|\right|}{\left|\left|b\right|\right|}$$

Apply the theorem above to Equation (19), we have,

$$\frac{\left|\left|\delta f\right|\right|}{k(A)\left|\left|f\right|\right|} \le \frac{\left|\left|\delta u\right|\right|}{\left|\left|u\right|\right|} \le \frac{k(A)\left|\left|\delta f\right|\right|}{\left|\left|f\right|\right|}$$

where, k(A) is the condition number.

The error analysis is performed in the post-processing stage. For the simplicity of the calculation, it is assumed that:

$$|u^{\prime\prime\prime}(x^*)|\overline{max}|u^{\prime\prime\prime}(x)|$$

Table 3. Error analysis for linear elements

	$ \delta f $	$ \delta x $	$k(A) \delta f $	k(A)
	k(A) f	x	f	
Ex 1.	6.8341e-08	3.0954e-04	0.23353	584.56
Ex 2.	7.1111e-07	0.030277	0.1481	456.35

From Table 3 above, it is evident that when the condition number of matrix K is increased, then the relative error increases. In conclusion, two factors affect the error accuracy, which is Δx and condition number of matrix K.

4. CONCLUSIONS

In this study, *h*-adaptive and traditional finite element method that enables high accurate approximations of high gradient advection-dominated problems have been proposed. The *h*-adaptive quickly identifies the fine region and concentrates in that region by further subdivide the area without further increase the elements to get better accuracy. This has also saved time and computational stress and minimizes error compared to the traditional finite element method, which requires more time and computational stress to deal with the generated algebraic equation due to the many elements involved. This approach is recommended for general differential equations with high gradients.

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NOMENCLATURE

- *k* condition number
- f load vector
- *K* stiffness matrix
- h mesh D conductiv
- Dconductivitywweight function

Greek symbols

α high-gradient value